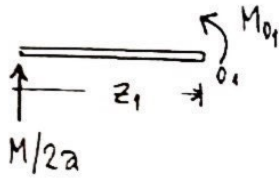


Şekildeki gıkmalı kırısin elastik eğri ifadesini Diferansiyel denklem yöntemiyle bulunuz.

Gözüm: Problemi üç bölgede incelemek uygundur. Önce mesnet tepkilerini hesaplayalım.

$$\Sigma M_B = 0 \rightarrow A_y \times 2a + M = 0 \quad A_y = M/2a \quad B_y = (2qa - \frac{M}{2a})$$

1. Bölge $0 \leq z_1 \leq a$



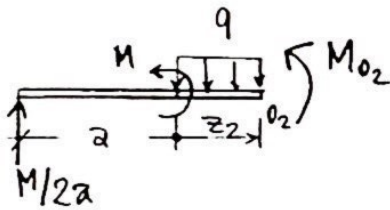
$$M_{01} = \frac{M}{2a} z_1$$

$$EI v_1'' = -\frac{M z_1}{2a}$$

$$EI v_1' = -\frac{M z_1^2}{4a} + C_1$$

$$EI v_1 = -\frac{M z_1^3}{12a} + C_1 z_1 + C_2$$

2. Bölge $0 \leq z_2 \leq a$



$$M_{02} = \frac{M}{2a} (z_2 + a) - M - \frac{q z_2^2}{2}$$

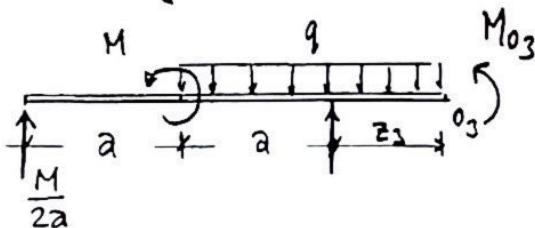
$$M_{02} = \frac{M z_2}{2a} - \frac{q z_2^2}{2} - \frac{M}{2}$$

$$EI v_2'' = -\frac{M}{2a} z_2 + \frac{q}{2} z_2^2 + \frac{M}{2}$$

$$EI v_2' = -\frac{M}{4a} z_2^2 + \frac{q}{6} z_2^3 + \frac{M}{2} z_2 + D_1$$

$$EI v_2 = -\frac{M}{12a} z_2^3 + \frac{q}{24} z_2^4 + \frac{M}{4} z_2^2 + D_1 z_2 + D_2$$

3. Bölge $0 \leq z_3 \leq a$



$$M_{03} = \frac{M}{2a} (z_3 + 2a) - M - q \frac{(z_3 + a)^2}{2} + (2qa - \frac{M}{2a}) z_3$$

$$M_{03} = -\frac{q}{2} z_3^2 + qa z_3 - \frac{qa^2}{2}$$

$$EI v_3'' = \frac{q}{2} z_3^2 - qa z_3 + \frac{qa^2}{2}$$

$$EI v_3' = \frac{q}{6} z_3^3 - \frac{qa}{2} z_3^2 + \frac{qa^2}{2} z_3 + E_1$$

$$EI v_3 = \frac{q}{24} z_3^4 - \frac{qa}{6} z_3^3 + \frac{qa^2}{4} z_3^2 + E_1 z_3 + E_2$$

Sınır koşulları: $v_1|_{z_1=0} = 0$, $v_2|_{z_2=a} = 0$, $v_3|_{z_3=0} = 0$

Süreklilik koşulları $v_1|_{z_1=a} = v_2|_{z_2=0}$, $v_1'|_{z_1=a} = v_2'|_{z_2=0}$, $v_2'|_{z_2=a} = v_3'|_{z_3=0}$

Sıradaki integrasyon sabitlerini belirleyelim.

$$v_1|_{z_1=0} = 0 \rightarrow C_2 = 0$$

$$v_1|_{z_1=a} = v_2|_{z_2=0} \rightarrow -\frac{Ma^2}{12} + C_1 a = D_2$$

$$v_1'|_{z_1=a} = v_2'|_{z_2=0} \rightarrow -\frac{Ma}{4} + C_1 = D_1$$

$$v_2|_{z_2=a} = 0 \rightarrow -\frac{Ma^2}{12} + \frac{qa^4}{24} + \frac{Ma^2}{4} + D_1 a + D_2 = 0$$

Bu denklemlerden C_1, D_1 ve D_2 çözülürse

$$C_1 = \frac{Ma}{12} - \frac{qa^3}{48} \quad D_1 = -\frac{Ma}{6} - \frac{qa^3}{48} \quad D_2 = -\frac{qa^4}{48}$$

bulunur. $v_3|_{z_3=0} = 0 \rightarrow E_2 = 0$ dir.

$$v_2'|_{z_2=a} = v_3'|_{z_3=0} \rightarrow -\frac{Ma}{4} + \frac{qa^3}{6} + \frac{Ma}{2} - \frac{Ma}{6} - \frac{qa^3}{48} = E_1$$

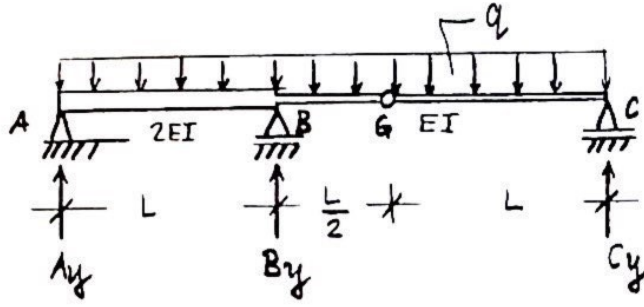
$$E_1 = \frac{Ma}{12} + \frac{7qa^3}{48}$$

Her bölge için elastik eğri ifadesi

$$EI v_1(z_1) = -\frac{M}{12a} z_1^3 + \left(\frac{Ma}{12} - \frac{qa^3}{48}\right) z_1$$

$$EI v_2(z_2) = -\frac{M}{12a} z_2^3 + \frac{q}{24} z_2^4 + \frac{M}{4} z_2^2 + \left(-\frac{Ma}{6} - \frac{qa^3}{48}\right) z_2 - \frac{qa^4}{48}$$

$$EI v_3(z_3) = \frac{q}{24} z_3^4 - \frac{qa}{6} z_3^3 + \frac{qa^2}{4} z_3^2 + \left(\frac{Ma}{12} + \frac{7qa^3}{48}\right) z_3$$



Şekildeki kiriş q yayılı yükü etkisindektir.

a) Elastik eğri ifadesini integrasyon yöntemi ile hesaplayınız.

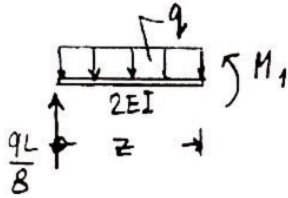
b) G mafsallındaki dönme sürekli-sızlığını hesaplayınız.

Gözüm: Mesnet tepkileri: $\sum M_{Gsol} = 0 \rightarrow C_y = \frac{1}{2} qL$

$$\sum M_A = 0 \rightarrow B_y \cdot L + 5 \frac{L}{2} C_y - q \frac{5L}{2} \cdot \frac{5L}{4} = 0 \rightarrow B_y = \frac{15}{8} qL$$

$$\sum Y = 0 \rightarrow A_y = \frac{5}{2} qL - \frac{19}{8} qL \quad A_y = \frac{qL}{8}$$

1. Bölge $0 \leq z \leq L$



$$M_1 = \frac{qLz}{8} - \frac{qz^2}{2}$$

$$2EI v_1'' = \frac{qz^2}{2} - \frac{qL}{8} z$$

$$2EI v_1' = \frac{qz^3}{6} - \frac{qL}{16} z^2 + C_1$$

$$2EI v_1 = \frac{qz^4}{24} - \frac{qL}{48} z^3 + C_1 z + C_2$$

Sınır koşullarımızı uygulayacak olursak

$$v_1(z) \Big|_{z=0} = 0 \rightarrow C_2 = 0$$

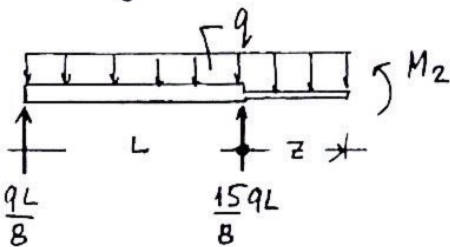
$$v_1'(z) \Big|_{z=L} = 0 \rightarrow \frac{qL^4}{24} - \frac{qL^4}{48} + C_1 L = 0 \rightarrow C_1 = -\frac{qL^3}{48}$$

1. Bölge için elastik eğri ifadesi

$$2EI v_1 = \frac{qz^4}{24} - \frac{qL}{48} z^3 - \frac{qL^3}{48} z$$

$$2EI v_1' = \frac{qz^3}{6} - \frac{qL}{16} z^2 - \frac{qL^3}{48}$$

2. Bölge $0 \leq z \leq L/2$



$$M_2 = -\frac{qz^2}{2} + qLz - \frac{3}{8} qL$$

$$EI v_2'' = \frac{qz^2}{2} - qLz + \frac{3}{8} qL$$

$$EI v_2' = \frac{qz^3}{6} - \frac{qL}{2} z^2 + \frac{3}{8} qL^2 z + D_1$$

$$EI v_2 = \frac{qz^4}{24} - \frac{qL}{6} z^3 + \frac{3}{16} qL^2 z^2 + D_1 z + D_2$$

2. bölgede sınır koşullarını yazarak olursak

$$v_2|_{z=0} = 0 \rightarrow D_2 = 0$$

süreklilik koşulu ise $v_1'(z)|_{z=L} = v_2'(z)|_{z=0}$

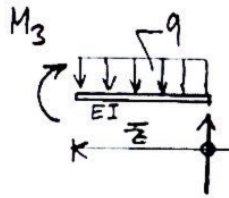
$$\frac{qL^3}{12} - \frac{qL^3}{32} - \frac{qL^3}{48} = D_1 \quad D_1 = \frac{qL^3}{32}$$

2. bölgede elastik eğri ifadesi

$$EI v_2 = \frac{qz^4}{24} - \frac{qL}{6} z^3 + \frac{3}{16} qL^2 z^2 + \frac{qL^3}{32} z$$

$$EI v_2' = \frac{qz^3}{6} - \frac{qL}{2} z^2 + \frac{3}{8} qL^2 z + \frac{qL^3}{32}$$

3. Bölge $0 \leq \bar{z} \leq L$



$$M_3 = \frac{1}{2} qL \bar{z} - \frac{1}{2} q \bar{z}^2$$

$$EI v_3''(\bar{z}) = \frac{q\bar{z}^2}{2} - \frac{qL}{2} \bar{z}$$

$$EI v_3'(\bar{z}) = \frac{q\bar{z}^3}{6} - \frac{qL}{4} \bar{z}^2 + E_1$$

$$EI v_3(\bar{z}) = \frac{q\bar{z}^4}{24} - \frac{qL}{12} \bar{z}^3 + E_1 \bar{z} + E_2$$

Sınır koşulu $v_3|_{\bar{z}=0} = 0 \rightarrow E_2 = 0$

Süreklilik koşulu $v_2|_{z=L/2} = v_3|_{\bar{z}=L}$

$$\frac{qL^4}{384} - \frac{qL^4}{48} + \frac{3}{64} qL^4 + \frac{qL^4}{64} = \frac{qL^4}{24} - \frac{qL^4}{12} + E_1 L \rightarrow E_1 = \frac{11}{128} qL^3$$

3. bölgede elastik eğri

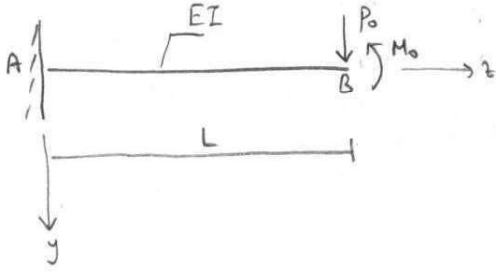
$$EI v_3(\bar{z}) = \frac{q\bar{z}^4}{24} - \frac{qL}{12} \bar{z}^3 + \frac{11}{128} qL^3 \bar{z}$$

G mafsalındaki dönme

$\bar{z}$ artım yönü z 'nin tersinde bu yüzden $v_3'(z) = -v_3'(\bar{z})$ olur.

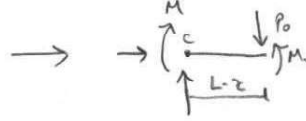
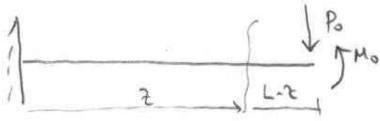
$$\Omega_G = -v_3'(\bar{z})|_{\bar{z}=L} - v_2'(z)|_{z=L/2} = -\frac{1}{384} \frac{qL^3}{EI} - \frac{11}{96} \frac{qL^3}{EI} = -\frac{15}{128} \frac{qL^3}{EI}$$

Uyg.-2:



Sekildeki konsol kırıgın açıklığı L, eğilme rijitliği EI sabittir. B uçundaki dönmenin sıfır olması için P_0 kuvveti ile eğilme momenti M_0 arasında sağlanılması gereken koşulu diferansiyel denklem yöntemiyle elde ediniz.

Çözüm:

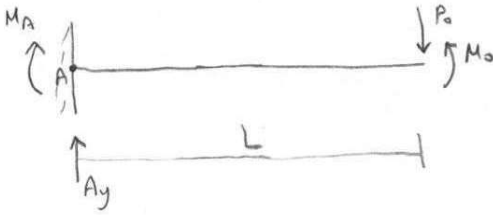


$$(+\sum M_c = 0)$$

$$M_0 - M - P_0(L-z) = 0$$

$$M = M_0 - P_0(L-z)$$

veya M kesit tensiyon değeri şu şekilde bulunabilir.



$$(+\sum M = 0)$$

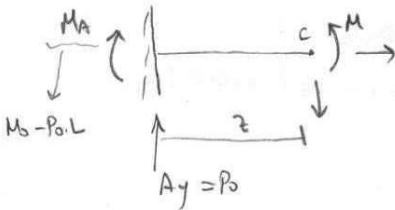
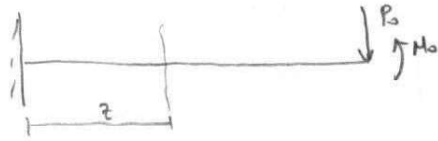
$$-M_A + M_0 - P_0 \cdot L = 0$$

$$M_A = M_0 - P_0 \cdot L$$

$$(+\sum F_y = 0)$$

$$A_y - P_0 = 0$$

$$A_y = P_0$$



$$(+\sum M_c = 0)$$

$$-(M_0 - P_0 \cdot L) - P_0 \cdot z + M = 0$$

$$M = M_0 - P_0(L-z)$$

→ aynı sonuç bulunur.

$$\frac{d^2v}{dz^2} = -\frac{M}{EI}$$

$$\frac{d^2v}{dz^2} = -\frac{(M_0 - P_0(L-z))}{EI} = -\frac{M_0}{EI} + \frac{P_0 \cdot L}{EI} - \frac{P_0 \cdot z}{EI} = \frac{1}{EI} (-M_0 + P_0 \cdot L - P_0 \cdot z)$$

$$\frac{dv}{dz} = \frac{1}{EI} \left(-M_0 z + P_0 \cdot L \cdot z - P_0 \frac{z^2}{2} \right) + C_1$$

$$V(z) = \frac{1}{EI} \left(-M_0 \frac{z^2}{2} + P_0 \cdot L \frac{z^2}{2} - P_0 \frac{z^3}{6} \right) + C_1 z + C_2$$

Sınır şartları:

$$z=0 \rightarrow V(0) = 0$$

$$\frac{1}{EI} \left(-M_0 \cdot 0 + P_0 \cdot L \cdot 0 - P_0 \cdot \frac{0}{6} \right) + C_1 \cdot 0 + C_2 = 0 \quad \boxed{C_2 = 0}$$

$$z=0 \rightarrow \varphi(0) = \frac{dv(0)}{dz} = v'(0) = 0$$

$$\frac{1}{EI} \left(-M_0 \cdot 0 + P_0 \cdot L \cdot 0 - P_0 \cdot \frac{0}{2} \right) + C_1 = 0 \quad \boxed{C_1 = 0}$$

$$V(z) = \frac{1}{EI} \left(-M_0 \frac{z^2}{2} + P_0 \cdot L \frac{z^2}{2} - P_0 \frac{z^3}{6} \right) \rightarrow \text{Elastik Eğri}$$

Burada $\rightarrow$ denge $\Rightarrow \varphi_B(L) = 0$ olması lazım

($z=L$) $\nearrow$

$$\frac{dv}{dz} = \varphi(z) = \frac{1}{EI} \left(-M_0 z + P_0 \cdot L \cdot z - P_0 \frac{z^2}{2} \right)$$

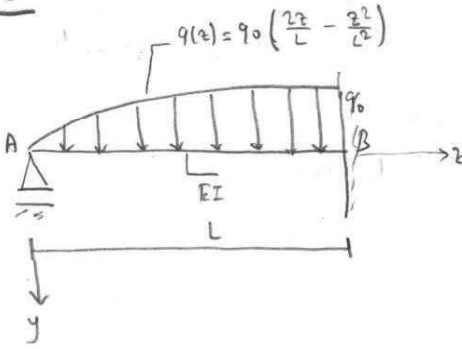
$$\varphi_B(L) = 0 \rightarrow \frac{1}{EI} \left(-M_0 \cdot L + P_0 \cdot L^2 - P_0 \frac{L^2}{2} \right) = 0$$

$$-M_0 \cdot L + \frac{P_0 \cdot L^2}{2} = 0$$

$$\boxed{M_0 = \frac{1}{2} P_0 \cdot L}$$

$\rightarrow$ olmasıdır.

vyg.-3!



Sekilde L uzunluğunda kırıktan parabolik bir yük etkiliyor. Diferansiyel denklemler yöntemiyle elde edilecek elastik eğri ifadesinden faydalabırda mesnet tepkilerini bulunuz.

Çözüm:

Dördüncü dereceden diferansiyel denklemlerle ilgili yük ifadesi yerleştirilerek dört kez integre edilir.

$$\frac{d^4v}{dz^4} = \frac{q_z}{EI} = \frac{1}{EI} \cdot q_0 \left(\frac{z}{L} - \frac{z^2}{L^2} \right)$$

$$\frac{d^3v}{dz^3} = \frac{1}{EI} \left(\frac{q_0 z}{L} - \frac{q_0 z^2}{2L^2} \right) + C_1$$

$$\frac{d^2v}{dz^2} = \frac{1}{EI} \left(\frac{q_0 z^2}{2L} - \frac{q_0 z^3}{6L^2} \right) + C_1 z + C_2$$

$$\frac{dv}{dz} = \frac{1}{EI} \left(\frac{q_0 z^3}{6L} - \frac{q_0 z^4}{24L^2} \right) + C_1 \frac{z^2}{2} + C_2 z + C_3$$

$$v(z) = \frac{1}{EI} \left(\frac{q_0 z^4}{24L} - \frac{q_0 z^5}{120L^2} \right) + C_1 \frac{z^3}{6} + C_2 \frac{z^2}{2} + C_3 z + C_4$$

Sınır Şartları:

$$z=0 \rightarrow v(0)=0 \rightarrow \frac{1}{EI} \left(\frac{q_0 \cdot 0}{24L} - \frac{q_0 \cdot 0}{120L^2} \right) + C_1 \cdot \frac{0}{6} + C_2 \cdot \frac{0}{2} + C_3 \cdot 0 + C_4 = 0 \quad \boxed{C_4=0}$$

$$z=0 \rightarrow \frac{d^2v}{dz^2} = 0 \rightarrow \frac{1}{EI} \left(\frac{q_0 \cdot 0}{2L} - \frac{q_0 \cdot 0}{6L^2} \right) + C_1 \cdot 0 + C_2 = 0 \quad \boxed{C_2=0}$$

$$z=L \rightarrow \frac{dv}{dz} = 0 \rightarrow \frac{1}{EI} \left(\frac{q_0 \cdot L^4}{24L} - \frac{q_0 \cdot L^5}{120L^2} \right) + C_1 \cdot \frac{L^2}{2} + C_3 = 0$$

$$\frac{1}{EI} \left(\frac{q_0 \cdot L^3}{24} - \frac{q_0 \cdot L^3}{60} \right) + C_1 \cdot \frac{L^2}{2} + C_3 = 0$$

$$\boxed{\frac{1}{EI} \left(\frac{q_0 L^3}{24} \right) + C_1 \cdot \frac{L^2}{2} + C_3 = 0} \quad \text{--- (I)}$$

$$z=L \quad v(L)=0 \quad \rightarrow \quad \frac{1}{EI} \left(\frac{q_0 L^5}{60L} - \frac{q_0 L^6}{360L^2} \right) + c_1 \frac{L^3}{6} + c_2 \frac{L^2}{2} + c_3 L + c_4 = 0$$

$$\frac{1}{EI} \left(\frac{q_0 L^4}{60} - \frac{q_0 L^4}{360} \right) + c_1 \frac{L^3}{6} + c_2 L = 0$$

$$\frac{1}{EI} \left(\frac{q_0 L^4}{72} \right) + c_1 \frac{L^3}{6} + c_3 L = 0$$

$$\boxed{\frac{1}{EI} \left(\frac{q_0 L^3}{72} \right) + c_1 \frac{L^2}{6} + c_3 = 0} \quad \text{--- (II)}$$

(I) ve (II) denklemleri birlikte gözölürse;

$$\frac{1}{EI} \left(\frac{q_0 L^3}{15} \right) + c_1 \frac{L^2}{2} + c_3 = 0 \quad \text{--- (I)}$$

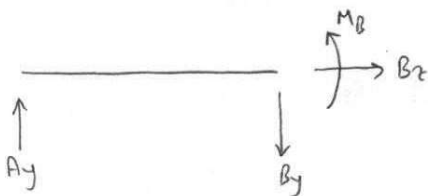
$$\frac{1}{EI} \left(\frac{q_0 L^3}{72} \right) + c_1 \frac{L^2}{6} + c_3 = 0 \quad \text{--- (II)}$$

$$\frac{1}{EI} \left(\frac{q_0 L^3}{15} - \frac{q_0 L^3}{72} \right) + c_1 \left(\frac{L^2}{2} - \frac{L^2}{6} \right) = 0 \quad \rightarrow \quad \boxed{c_1 = \frac{-19}{120EI} q_0 L}$$

c_1 değeri I veya II denkleminde yerine yazılır. ve c_3 bulunur.

$$\boxed{c_3 = \frac{1}{80EI} q_0 L^3}$$

$$\boxed{V(z) = \frac{1}{EI} \left(\frac{q_0}{60L} z^5 - \frac{q_0}{360L^2} z^6 + \frac{q_0 L^2}{80} z - \frac{19 q_0 L}{720} z^2 \right)} \quad \rightarrow \text{Elastik Eğri}$$



$$\frac{d^3 v}{dz^3} = - \frac{A_y}{EI} \quad \rightarrow \quad EI \frac{d^3 v}{dz^3} = -A_y$$

$$EI \left[\frac{1}{EI} \left(\frac{q_0 z^2}{2} - \frac{q_0 z^3}{3L^2} - \frac{19}{120} q_0 L \right) \right] = -A_y$$

$$z=0 \quad \rightarrow \quad EI \frac{d^3 v}{dz^3} = -A_y$$

$$\frac{q_0 \cdot 0}{L} - \frac{q_0 \cdot 0}{3L^2} - \frac{19}{120} q_0 L = -A_y$$

$$\boxed{A_y = \frac{19}{120} q_0 L}$$

$$z=L \rightarrow \frac{d^2v}{dz^2} = -\frac{M_B}{EI} \Rightarrow EI \frac{d^2v}{dz^2} = -M_B$$

$$EI \left[\frac{1}{EI} \left(\frac{q_0 \cdot L^3}{3L} - \frac{q_0 \cdot L^4}{12L^2} \right) - \frac{19}{120} q_0 \cdot L^2 \right] = -M_B$$

$$M_B = -\frac{11}{120} q_0 \cdot L^2$$

$$z=L \rightarrow \frac{d^3v}{dz^3} = -\frac{B_y}{EI} \Rightarrow EI \frac{d^3v}{dz^3} = -B_y$$

$$EI \left[\frac{1}{EI} \left(\frac{q_0 \cdot L^2}{2} - \frac{q_0 \cdot L^3}{3L^2} - \frac{19}{120} q_0 \cdot L \right) \right] = -B_y$$

$$B_y = -\frac{61}{120} q_0 \cdot L$$

Yataay denge dakteleni $\rightarrow \Sigma F_z = 0 \Rightarrow B_z = 0$

Tanımlar

$$\frac{dT}{dx} = -q \quad \Leftrightarrow T = -\int q dx$$

$$\frac{dM}{dx} = T \quad \Leftrightarrow M = \int T dx = -\int \int q dx^2$$

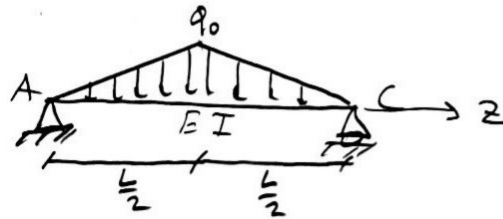
$$\frac{dQ}{dx} = Q \quad \Leftrightarrow Q = \int Q dx$$

$$K = \frac{1}{\rho} = \frac{dQ}{dx} = \frac{d^2 Q}{dx^2} = -\frac{M}{EI} \quad \Leftrightarrow Q = -\int \frac{M}{EI} dx^2 \quad \text{ve} \quad Q = -\int \frac{M}{EI} dx$$

$$\frac{d^3 Q}{dx^3} = \frac{d}{dx} \left(-\frac{M}{EI} \right) = -\frac{T}{EI} \quad ; \quad \frac{d^4 Q}{dx^4} = \frac{d}{dx} \left(-\frac{T}{EI} \right) = \frac{q}{EI}$$

SORU: Şekildeki basit kiriş simetrik üşgen yığılı yük etkisindedir. Diferansiyel denklem yöntemi ile elastik eğri denklemini yazınız. A noktasındaki dönme yi ve kırıste meydana gelen en büyük çökme yi hesaplayınız.

Çözüm: Kiriş simetrik olduğundan yarısı için hesap yapılabilir.

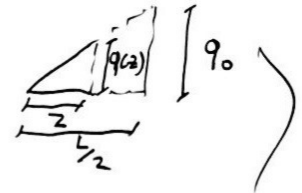


$$\Delta m = \frac{q_0}{\frac{L}{2}} = \frac{2q_0}{L}$$

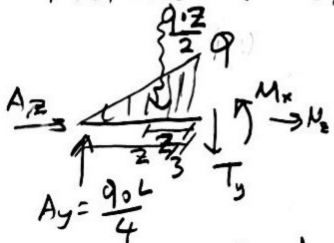
$$q = q(z) = m z + q_0 = \frac{2q_0}{L} z$$

(veya üşgen benzerliğine göre)

$$\frac{q(z)}{\frac{L}{2}} = \frac{q_0}{\frac{L}{2}}$$



moment fonksiyonu:



$$M_x = \frac{q_0 L}{4} z - \frac{q_0}{2} \cdot \frac{z}{3}$$

$$M_x = \frac{q_0 L}{4} z - \frac{q_0}{3L} z^3$$

diferansiyel denklem:

$$-EI_x v''(z) = \frac{q_0 L}{4} z - \frac{q_0}{3L} z^3$$

$$-EI_x v'(z) = \frac{q_0 L}{8} z^2 - \frac{q_0}{12L} z^4 + C_1$$

$$-EI_x v(z) = \frac{q_0 L}{24} z^3 - \frac{q_0}{60L} z^5 + C_1 z + C_2$$

Sınır koşulları:

$$-EI_x \psi(0) = 0 \Rightarrow C_2 = 0$$

$$-EI_x \psi'(\frac{L}{2}) = 0 \Rightarrow \frac{q_0 L}{8} (\frac{L}{2})^2 - \frac{q_0}{12L} (\frac{L}{2})^4 + C_1 = 0$$

$$C_1 = \frac{q_0 L^3}{12 \cdot 16} - \frac{q_0 L^3}{32} = -\frac{5q_0 L^3}{192}$$

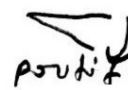
Elastik eğri ve dönme esitlikleri ($0 \leq z \leq \frac{1}{2}L$)

$$\psi'(z) = -\frac{1}{EI_x} \left[\frac{q_0 L}{8} z^2 - \frac{q_0}{12L} z^4 - \frac{5q_0 L^3}{192} \right]$$

$$\psi(z) = -\frac{1}{EI_x} \left[\frac{q_0 L}{24} z^3 - \frac{q_0}{60L} z^5 - \frac{5q_0 L^3}{192} z \right]$$

A noktasındaki dönme:

$$\theta_A = \theta(0) = \psi'(0) = \frac{5q_0 L^3}{192 EI}$$

Dönme yönü 

Maksimum sehim:

$\psi(z)$ 'nin ekstremum noktası aranmalıdır.

$$\psi'(z) = \theta(z) = 0$$

$$-\frac{q_0}{12L} z^4 + \frac{q_0 L}{8} z^2 - \frac{5q_0 L^3}{192} = 0$$

$$z^2 = t \text{ olsun}$$

$$t_{1,2} = \frac{-\frac{q_0 L}{8} \pm \sqrt{\frac{q_0^2 L^2}{64} - \frac{5q_0^2 L^3}{3 \cdot 192L}}}{-\frac{q_0}{6L}} = \frac{\frac{3L^2}{4} \pm \frac{6L}{8} \sqrt{\frac{9q_0^2 L^2}{576} - \frac{5q_0^2 L^2}{576}}}{-\frac{q_0}{6L}} = \frac{\frac{3L^2}{4} \pm \frac{12L^2}{24}}{-\frac{q_0}{6L}}$$

$$t_{1,2} = \frac{\frac{3L^2}{4} \pm \frac{2L^2}{4}}{-\frac{q_0}{6L}} = \frac{L^2}{4} (3 \mp 2)$$

$$z_{1,2,3,4} = \pm \sqrt{t_{1,2}} = \pm \frac{L\sqrt{3 \mp 2}}{2}$$

$$0 \leq z \leq \frac{L}{2} \Rightarrow z_1 = \frac{L\sqrt{5}}{2} \quad z_2 = \frac{L}{2} \quad z_3 = -\frac{L\sqrt{5}}{2} \quad z_4 = -\frac{L}{2}$$

$$\psi(z_2 = \frac{L}{2}) = -\frac{1}{EI_x} \left[\frac{q_0 L}{24} (\frac{L}{2})^3 - \frac{q_0}{60L} (\frac{L}{2})^5 - \frac{5q_0 L^3}{192} \cdot \frac{L}{2} \right] = \frac{q_0 L^4}{120 EI_x}$$